

# Equation Sheet

## 1 Newtons second law and equations of kinematics

The net force  $\mathbf{F}$  on a particle of mass  $m$  is given by

$$\sum_i \mathbf{F}_i = \mathbf{F} = m\mathbf{a} = m \frac{d\mathbf{v}}{dt}. \quad (1)$$

For a system of particles

$$\mathbf{F} = m\mathbf{a}_G = m\dot{\mathbf{v}}_G. \quad (2)$$

## 2 Linear and curvilinear motion

### 2.1 Motion along a path

$$\begin{aligned} s, \\ \dot{s} = v, \\ \ddot{s} = \dot{v} = a, \\ a ds = v dv. \end{aligned} \quad (3)$$

### 2.2 Angular motion relations

$$\begin{aligned} \theta, \\ \dot{\theta} = \omega, \\ \ddot{\theta} = \dot{\omega} = \alpha, \\ \alpha d\theta = \omega d\omega. \end{aligned} \quad (4)$$

### 2.3 Projectile motion

$$\begin{aligned}
 a_x &= 0, & a_y &= -g, \\
 v_x &= (v_x)_o, & v_y &= (v_y)_o - gt, \\
 x &= x_o + (v_x)_o t, & y &= y_o + (v_y)_o t - g \frac{t^2}{2}, \\
 v_y^2 &= (v_y)_o^2 - 2g(y - y_o).
 \end{aligned} \tag{5}$$

## 3 Vector relations

### 3.1 Cartesian

$$\begin{aligned}
 \mathbf{r} &= x\mathbf{i} + y\mathbf{j} + z\mathbf{k}, \\
 \dot{\mathbf{r}} &= \mathbf{v} = \dot{x}\mathbf{i} + \dot{y}\mathbf{j} + \dot{z}\mathbf{k}, \\
 \ddot{\mathbf{r}} &= \dot{\mathbf{v}} = \mathbf{a} = \ddot{x}\mathbf{i} + \ddot{y}\mathbf{j} + \ddot{z}\mathbf{k}.
 \end{aligned} \tag{6}$$

### 3.2 Normal and tangential

$$\begin{aligned}
 \mathbf{v} &= v\mathbf{e}_t, \\
 \dot{\mathbf{v}} &= \mathbf{a} = \dot{v}\mathbf{e}_t + \frac{v^2}{\rho}\mathbf{e}_n.
 \end{aligned} \tag{7}$$

### 3.3 Polar coordinates

$$\begin{aligned}
 \mathbf{r} &= r\mathbf{e}_r, \\
 \dot{\mathbf{r}} &= \mathbf{v} = \dot{r}\mathbf{e}_r + r\dot{\theta}\mathbf{e}_\theta + \dot{z}\mathbf{k}, \\
 \ddot{\mathbf{r}} &= \dot{\mathbf{v}} = \mathbf{a} = (\ddot{r} - r\dot{\theta}^2)\mathbf{e}_r + (r\ddot{\theta} + 2\dot{r}\dot{\theta})\mathbf{e}_\theta + \ddot{z}\mathbf{k}.
 \end{aligned} \tag{8}$$

Note that  $\alpha$  is often used to represent  $\ddot{\theta}$  and  $\omega$  is substituted for  $\dot{\theta}$ .

## 4 Relative velocities and accelerations (vector addition)

Position of A relative to B is denoted  $\mathbf{r}_{A/B}$

Similar vector addition rules apply for velocity and acceleration.

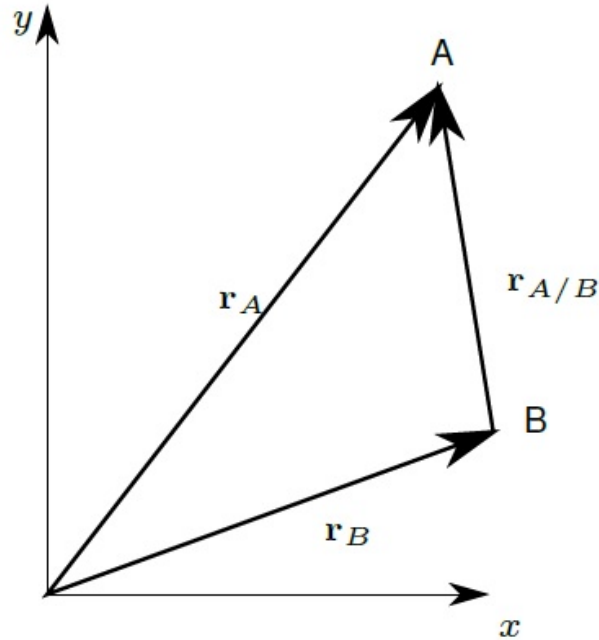


Fig. 8: Rule of vector addition.

$$\begin{aligned}
 \mathbf{r}_A &= \mathbf{r}_B + \mathbf{r}_{A/B}, \\
 \mathbf{v}_A &= \mathbf{v}_B + \mathbf{v}_{A/B}, \\
 \mathbf{a}_A &= \mathbf{a}_B + \mathbf{a}_{A/B}.
 \end{aligned}
 \tag{9}$$

Relative to a point B the position, velocity, and acceleration of a point A can be determined from

$$\begin{aligned}
 \mathbf{v}_{A/B} &= \boldsymbol{\omega}_{A/B} \times \mathbf{r}_{A/B}, \\
 \mathbf{a}_{A/B} &= (\mathbf{a}_{A/B})_n + (\mathbf{a}_{A/B})_t, \\
 (\mathbf{a}_{A/B})_t &= \boldsymbol{\alpha}_{A/B} \times \mathbf{r}_{A/B}, \\
 (\mathbf{a}_{A/B})_n &= \boldsymbol{\omega}_{A/B} \times (\boldsymbol{\omega}_{A/B} \times \mathbf{r}_{A/B}).
 \end{aligned}
 \tag{10}$$

## 5 Work

$$dU = \mathbf{F} \cdot d\mathbf{r} = F dr \cos(\gamma) = F_t ds. \tag{11}$$

$$U_{1-2} = \int_1^2 \mathbf{F} \cdot d\mathbf{r} = \int_1^2 F_t ds \tag{12}$$

For an external force at angle  $\gamma$  with linear translation in direction  $dx$

$$U_{1-2} = \int_1^2 \mathbf{F} \cdot d\mathbf{r} = \int_1^2 F \cos(\gamma) dx \quad (13)$$

Work associated with a spring (a negative value indicates work done on the spring)

$$U_{1-2} = -\frac{1}{2}k(x_2^2 - x_1^2). \quad (14)$$

Work associated with gravity for small relative distances

$$U_{1-2} = -mg(y_2 - y_1). \quad (15)$$

Work associated with friction

$$U_{1-2} = -\mu F_N |x_2 - x_1|. \quad (16)$$

### 5.1 Work-energy equation

$$T_1 + U_{1-2} = T_2. \quad (17)$$

### 5.2 Work-energy equation(potential)

$$T_1 + V_1 + U'_{1-2} = T_2 + V_2, \quad (18)$$

$$V = V_g + V_e, \quad (19)$$

or

$$U'_{1-2} = \Delta T + \Delta V. \quad (20)$$

### 5.3 Potential energy

Stored.

Gravitational

$$V_g = mgh. \quad (21)$$

Elastic (linear spring)

$$V_e = \frac{1}{2}kx^2. \quad (22)$$

For a system of particles

$$T = \frac{1}{2}mv_G^2 + \frac{1}{2} \sum_i m_i |\dot{\mathbf{r}}_i|^2. \quad (23)$$

For rigid body, plane motion

$$T = \frac{1}{2}mv_G^2 + \frac{1}{2}I_G\omega^2, \quad (24)$$

$$= \frac{1}{2}I_O\omega^2. \quad (25)$$

Equation 25 is for rotation about a fixed axis.

Force on an object in the field of scalar potential

$$\mathbf{F} = -\Delta V(x, y, z), \quad (26)$$

$$\Delta = \mathbf{i} \frac{\partial}{\partial x} + \mathbf{j} \frac{\partial}{\partial y} + \mathbf{k} \frac{\partial}{\partial z} \quad (27)$$

## 6 Momentum and impulse equations

### 6.1 Distance to center of mass

$$\mathbf{r}_G = \frac{\sum m_i \mathbf{r}_i}{\sum m_i}, \quad (28)$$

or

$$m\mathbf{r}_G = \sum m_i \mathbf{r}_i. \quad (29)$$

### 6.2 Linear momentum and impulse

$$\mathbf{G}_1 + \int_{t_1}^{t_2} \sum \mathbf{F} dt = \mathbf{G}_2. \quad (30)$$

For a system of particles

$$\mathbf{G} = m\mathbf{v}_G. \quad (31)$$

and

$$\sum \mathbf{F} = \dot{\mathbf{G}}. \quad (32)$$

### 6.3 Moment of momentum and angular impulse

Angular momentum and moment of momentum are the same thing.

For a system of particles measured from a fixed point 'O'

$$\mathbf{H}_O = \sum \mathbf{r}_i \times m_i \mathbf{v}_i, \quad (33)$$

$$\sum \mathbf{M}_O = \sum \mathbf{r} \times \mathbf{F} = \dot{\mathbf{H}}_O. \quad (34)$$

### 6.4 Angular impulse equation

$$\mathbf{H}_{O_1} + \int_{t_1}^{t_2} \sum \mathbf{M}_O dt = \mathbf{H}_{O_2}. \quad (35)$$

### 6.5 About center of mass

$$\mathbf{H}_G = \sum_i \boldsymbol{\rho}_i \times m_i \mathbf{v}_i. \quad (36)$$

$$\sum \mathbf{M}_G = \dot{\mathbf{H}}_G. \quad (37)$$

### 6.6 About arbitrary point P

$$\mathbf{H}_P = \mathbf{H}_G + \boldsymbol{\rho}_{G/P} \times m \mathbf{v}_G. \quad (38)$$

$$\sum \mathbf{M}_P = \dot{\mathbf{H}}_G + \boldsymbol{\rho}_{G/P} \times m \mathbf{a}_G. \quad (39)$$

Relative moment of momentum about P is

$$(\mathbf{H}_P)_{\text{rel}} = \mathbf{H}_G + \boldsymbol{\rho}_{G/P} \times m \mathbf{v}_{P/G}. \quad (40)$$

$$\sum \mathbf{M}_P = (\dot{\mathbf{H}}_P)_{\text{rel}} + \boldsymbol{\rho}_{G/P} \times m \mathbf{a}_P. \quad (41)$$

or, equivalently

$$\sum \mathbf{M}_P = \dot{\mathbf{H}}_G + \boldsymbol{\rho}_{G/P} \times m \mathbf{a}_G. \quad (42)$$

### 6.7 Rigid body moments of momentum

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In addition to  $\mathbf{F} = m\mathbf{a}_G$ ,

$$\sum \mathbf{M}_G = \dot{\mathbf{H}}_G. \quad (43)$$

$$\sum \mathbf{M}_P = \dot{\mathbf{H}}_G + \boldsymbol{\rho}_{G/P} \times m\mathbf{a}_G. \quad (44)$$

$$\sum \mathbf{M}_P = I_P\boldsymbol{\alpha} + \boldsymbol{\rho}_{G/P} \times m\mathbf{a}_P. \quad (45)$$

or, equivalently

$$\sum \mathbf{M}_P = I_G\boldsymbol{\alpha} + \boldsymbol{\rho}_{G/P} \times m\mathbf{a}_G. \quad (46)$$

### 6.8 Planar rigid bodies

All moments and rotation axes share a common unit vector direction.

$$\sum M_G = I_G\alpha, \quad (47)$$

$$\sum M_O = I_O\alpha, \quad (48)$$

$$\sum M_P = I_G\alpha + ma_Gd, \quad (49)$$

$$\sum M_P = I_P\alpha + ma_Pd. \quad (50)$$

If  $P$  is not accelerating  $P$  becomes  $O$

$$d = \frac{|\boldsymbol{\rho} \times \mathbf{a}|}{a}. \quad (51)$$

$$H_o = I_G\omega + mv_Gd = I_O\omega. \quad (52)$$

### 7 Second moment of mass about a centroidal axis

$$I_G = \int^{\text{Vol.}} r^2 dm. \quad (53)$$

Parallel axis theorem

$$I_P = I_G + mL^2. \quad (54)$$

$P$  refers to a point that is fixed relative to the rigid body.

## 8 Impact

### 8.1 Coefficient of restitution

Linear impact

$$e = \frac{v'_2 - v'_1}{v_1 - v_2}. \quad (55)$$

### 8.2 Coefficient of restitution

Oblique impact.

$$e = \frac{(v'_2)_n - (v'_1)_n}{(v_1)_n - (v_2)_n}. \quad (56)$$

## 9 Rotation about a fixed axis

### 9.1 In vector form

$$\mathbf{v} = \boldsymbol{\omega} \times \mathbf{R}, \quad (57)$$

$$\mathbf{a} = \dot{\mathbf{v}} = \boldsymbol{\omega} \times \mathbf{v} + \dot{\boldsymbol{\omega}} \times \mathbf{R}, \quad (58)$$

$$= \boldsymbol{\omega} \times (\boldsymbol{\omega} \times \mathbf{R}) + \boldsymbol{\alpha} \times \mathbf{R} = \mathbf{a}_n + \mathbf{a}_t. \quad (59)$$

### 9.2 Scalar form

Constant  $R$ .

$$v = R\omega, \quad (60)$$

$$a_n = R\omega^2 = \frac{v^2}{R} = v\omega, \quad (61)$$

$$a_t = R\alpha. \quad (62)$$

## 10 Motion relative to rotating axes

$$\mathbf{v}_A = \mathbf{v}_B + \boldsymbol{\omega} \times \mathbf{r} + \mathbf{v}_{rel}, \quad (63)$$

$$\mathbf{a}_A = \mathbf{a}_B + \dot{\boldsymbol{\omega}} \times \mathbf{r} + \boldsymbol{\omega} \times (\boldsymbol{\omega} \times \mathbf{r}) + 2\boldsymbol{\omega} \times \mathbf{v}_{rel} + \mathbf{a}_{rel}. \quad (64)$$



## 11- Laplace Table

	$f(t)$	$F(s)$
1	Unit Impulse $\delta(t)$	1
2	Unit Step $u(t)$	$\frac{1}{s}$
3	Unit Ramp $r(t)$	$\frac{1}{s^2}$
4	$\frac{t^{n-1}}{(n-1)!}, n = 1, 2, 3 \dots$	$\frac{1}{s^n}$
5	$t^n, n = 1, 2, 3 \dots$	$\frac{n!}{s^{n+1}}$
6	$e^{-at}$	$\frac{1}{s+a}$
7	$te^{-at}$	$\frac{1}{(s+a)^2}$
8	$\frac{t^{n-1}}{(n-1)!} e^{-at}, n = 1, 2, 3 \dots$	$\frac{1}{(s+a)^n}$
9	$t^n e^{-at}, n = 1, 2, 3 \dots$	$\frac{n!}{(s+a)^{n+1}}$
10	$\sin(\omega t)$	$\frac{\omega}{s^2 + \omega^2}$
11	$\cos(\omega t)$	$\frac{s}{s^2 + \omega^2}$
12	$\sinh(\omega t)$	$\frac{\omega}{s^2 - \omega^2}$
13	$\cosh(\omega t)$	$\frac{s}{s^2 - \omega^2}$
14	$\frac{1}{a}(1 - e^{-at})$	$\frac{1}{s(s+a)}$
15	$\frac{1}{b-a}(e^{-at} - e^{-bt})$	$\frac{1}{(s+a)(s+b)}$
16	$\frac{1}{b-a}(be^{-bt} - ae^{-at})$	$\frac{s}{(s+a)(s+b)}$
17	$\frac{1}{ab}[1 + \frac{1}{a-b}(be^{-at} - ae^{-bt})]$	$\frac{1}{s(s+a)(s+b)}$
18	$\frac{1}{a^2}(1 - e^{-at} - ate^{-at})$	$\frac{1}{s(s+a)^2}$

19	$\frac{1}{a^2}(at - 1 + e^{-at})$	$\frac{1}{s^2(s+a)}$
20	$e^{-at} \sin(\omega t)$	$\frac{\omega}{(s+a)^2 + \omega^2}$
21	$e^{-at} \cos(\omega t)$	$\frac{s+a}{(s+a)^2 + \omega^2}$
22	$\frac{\omega_n}{\sqrt{1-\zeta^2}} e^{-\zeta\omega_n t} \sin(\omega_n \sqrt{1-\zeta^2} t)$	$\frac{\omega_n^2}{s^2 + 2\zeta\omega_n s + \omega_n^2}$
23	$-\frac{1}{\sqrt{1-\zeta^2}} e^{-\zeta\omega_n t} \sin(\omega_n \sqrt{1-\zeta^2} t - \phi)$ $\phi = \tan^{-1}\left(\frac{\sqrt{1-\zeta^2}}{\zeta}\right)$	$\frac{s}{s^2 + 2\zeta\omega_n s + \omega_n^2}$
24	$1 - \frac{1}{\sqrt{1-\zeta^2}} e^{-\zeta\omega_n t} \sin(\omega_n \sqrt{1-\zeta^2} t + \phi)$ $\phi = \tan^{-1}\left(\frac{\sqrt{1-\zeta^2}}{\zeta}\right)$	$\frac{\omega_n^2}{s(s^2 + 2\zeta\omega_n s + \omega_n^2)}$
25	$1 - \cos(\omega t)$	$\frac{\omega^2}{s(s^2 + \omega^2)}$
26	$\omega t - \sin(\omega t)$	$\frac{\omega^3}{s^2(s^2 + \omega^2)}$
27	$\sin(\omega t) - \omega t \cos(\omega t)$	$\frac{2\omega^3}{(s^2 + \omega^2)^2}$
28	$\frac{1}{2\omega} t \sin(\omega t)$	$\frac{s}{(s^2 + \omega^2)^2}$
29	$t \cos(\omega t)$	$\frac{s^2 - \omega^2}{(s^2 + \omega^2)^2}$
30	$\frac{1}{\omega_2^2 - \omega_1^2} [\cos(\omega_1 t) - \cos(\omega_2 t)], \omega_1^2 \neq \omega_2^2$	$\frac{s}{(s^2 + \omega_1^2)(s^2 + \omega_2^2)}$
31	$\frac{1}{2\omega} [\sin(\omega t) + \omega t \cos(\omega t)]$	$\frac{s^2}{(s^2 + \omega^2)^2}$

**TABLE 2-2** Properties of Laplace Transforms

1	$\mathcal{L}[Af(t)] = AF(s)$
2	$\mathcal{L}[f_1(t) \pm f_2(t)] = F_1(s) \pm F_2(s)$
3	$\mathcal{L}_{\pm}\left[\frac{d}{dt}f(t)\right] = sF(s) - f(0_{\pm})$
4	$\mathcal{L}_{\pm}\left[\frac{d^2}{dt^2}f(t)\right] = s^2F(s) - sf(0_{\pm}) - \dot{f}(0_{\pm})$
5	$\mathcal{L}_{\pm}\left[\frac{d^n}{dt^n}f(t)\right] = s^nF(s) - \sum_{k=1}^n s^{n-k} f^{(k-1)}(0_{\pm})$ where $f^{(k-1)}(t) = \frac{d^{k-1}}{dt^{k-1}}f(t)$
6	$\mathcal{L}_{\pm}\left[\int f(t) dt\right] = \frac{F(s)}{s} + \frac{[\int f(t) dt]_{t=0_{\pm}}}{s}$
7	$\mathcal{L}_{\pm}\left[\iint f(t) dt dt\right] = \frac{F(s)}{s^2} + \frac{[\int f(t) dt]_{t=0_{\pm}}}{s^2} + \frac{[\iint f(t) dt dt]_{t=0_{\pm}}}{s}$
8	$\mathcal{L}_{\pm}\left[\int \cdots \int f(t)(dt)^n\right] = \frac{F(s)}{s^n} + \sum_{k=1}^n \frac{1}{s^{n-k+1}} \left[\int \cdots \int f(t)(dt)^k\right]_{t=0_{\pm}}$
9	$\mathcal{L}\left[\int_0^t f(t) dt\right] = \frac{F(s)}{s}$
10	$\int_0^{\infty} f(t) dt = \lim_{s \rightarrow 0} F(s)$ if $\int_0^{\infty} f(t) dt$ exists
11	$\mathcal{L}[e^{-at}f(t)] = F(s + a)$
12	$\mathcal{L}[f(t - \alpha)1(t - \alpha)] = e^{-\alpha s}F(s) \quad \alpha \geq 0$
13	$\mathcal{L}[tf(t)] = -\frac{dF(s)}{ds}$
14	$\mathcal{L}[t^2f(t)] = \frac{d^2}{ds^2}F(s)$
15	$\mathcal{L}[t^n f(t)] = (-1)^n \frac{d^n}{ds^n}F(s) \quad n = 1, 2, 3, \dots$
16	$\mathcal{L}\left[\frac{1}{t}f(t)\right] = \int_s^{\infty} F(s) ds$ if $\lim_{t \rightarrow 0} \frac{1}{t}f(t)$ exists
17	$\mathcal{L}\left[f\left(\frac{t}{a}\right)\right] = aF(as)$

## 12- Quadratic Equation

$$As^2 + Bs + C = 0$$

$$s_{1,2} = \frac{-B \pm \sqrt{B^2 - 4AC}}{2A}$$

## 13- Newton's Second Law

Rectilinear motion in the  $x$  – direction

$$\sum f_x = m\ddot{x}$$

Rotation of a rigid body about pinned point  $O$

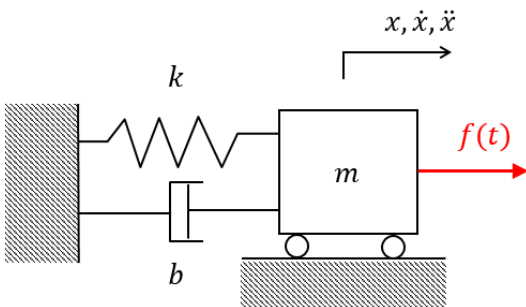
$$\sum m_o = J_o\ddot{\theta}$$

## 14. Initial Value Theorem and Final Value Theorem

$$\lim_{t \rightarrow 0} (x(t)) = \lim_{s \rightarrow \infty} (sX(s))$$

$$\lim_{t \rightarrow \infty} (x(t)) = \lim_{s \rightarrow 0} (sX(s))$$

## 15. Equation of Motion and Transfer Function for a Damped Harmonic Oscillator



$$\sum f_x = m\ddot{x}$$

$$m\ddot{x} + b\dot{x} + kx = f(t)$$

$$\frac{X(s)}{F(s)} = \frac{1}{ms^2 + bs + k}$$

## 17. Definition of Natural Frequency, Damping Ratio, and Damped Natural Frequency

Given a second order differential equation of motion  $A\ddot{x} + B\dot{x} + Cx = g(t)$  where  $A$ ,  $B$  and  $C$  are constants,  $x(t)$  is the dependent variable and  $t$  is the independent variable, the natural frequency, damping ratio and damped natural frequency can be defined.

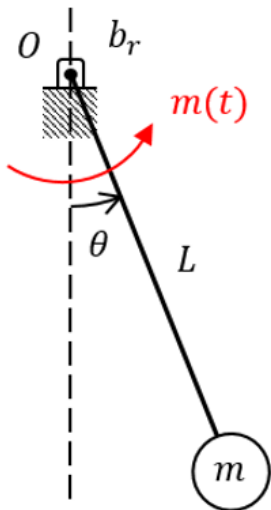
$$\omega_n = \sqrt{\frac{C}{A}}$$

$$\zeta = \frac{B}{2\sqrt{AC}} = \frac{B}{2A\omega_n}$$

$$\omega_d = \omega_n \sqrt{1 - \zeta^2}$$

$\zeta = 0$	Undamped, Oscillates Indefinitely, Purely Imaginary Roots
$\zeta < 1$	Underdamped, Exponentially Decaying Oscillation, Complex Roots
$\zeta = 1$	Critically Damped, No Oscillation, Repeated Real Roots
$\zeta > 1$	Overdamped, No Oscillation, Distinct Real Roots

## 18. Simple pendulum with rotational damping and input moment



Nonlinear Equation of Motion

$$mL^2\ddot{\theta} + b_r\dot{\theta} + mgL \sin \theta = m(t)$$

Linearized Equation of Motion

$$mL^2\ddot{\theta} + b_r\dot{\theta} + mgL\theta = m(t)$$

Transfer Function

$$\frac{\Theta(s)}{M(s)} = \frac{1}{mL^2s^2 + b_r s + mgL}$$

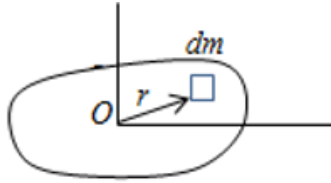
Natural Frequency and Damping Ratio

$$\omega_n = \sqrt{\frac{g}{L}}$$

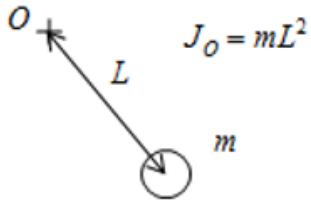
$$\zeta = \frac{b_r}{2mL^2\omega_n}$$

## 19. Mass moments of inertia

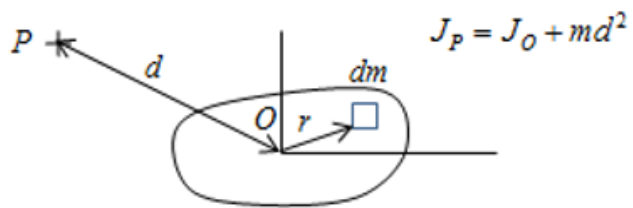
$$J_O = \int_V r^2 dm$$



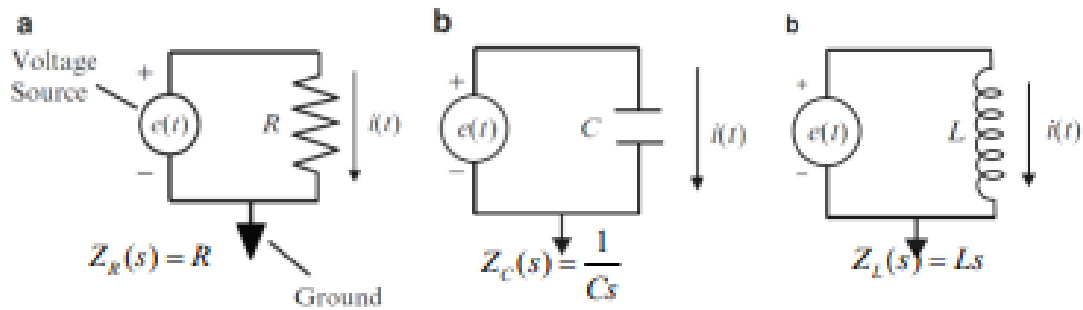
*Point Mass*



*Parallel axis theorem*



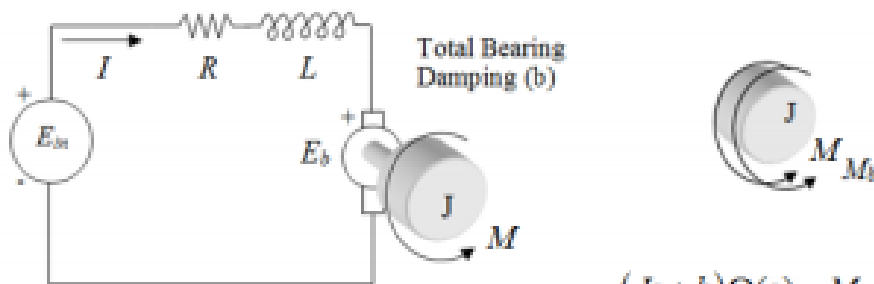
## Impedances of passive electrical circuit elements



## Transfer function of a permanent magnet DC-motor (Unloaded)

$$M = KI \quad (1)$$

$$E_b = K_b \Omega \quad (2)$$



$$E_m(s) - IR - ILs - E_b(s) = 0 \quad (3)$$

$$\frac{\Omega(s)}{E_m(s)} = \frac{K}{JLs^2 + (JR + bL)s + bR + KK_b}$$

$$\tau_{elec} \equiv \frac{L}{R}$$

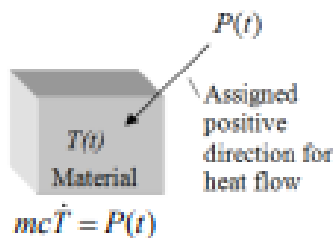
$$\tau_{mech} \equiv \frac{RJ}{KK_b}$$

## Loaded permanent magnet DC-motor (steady state with load $M_l$ ) – Power and Torque Curves

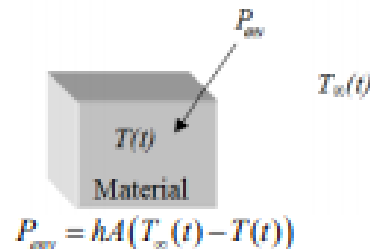
$$M_l = \frac{K}{R} V_o - \left( \frac{bR + KK_b}{R} \right) \omega_{ss}$$

$$P = M_l \omega_{ss} = \frac{KV_o}{R} \omega_{ss} - \frac{(KK_b + bR)}{R} \omega_{ss}^2$$

## Thermal Element



## Environmental Heat Exchange



$$\tau_{thermal} = \frac{mc}{hA}$$